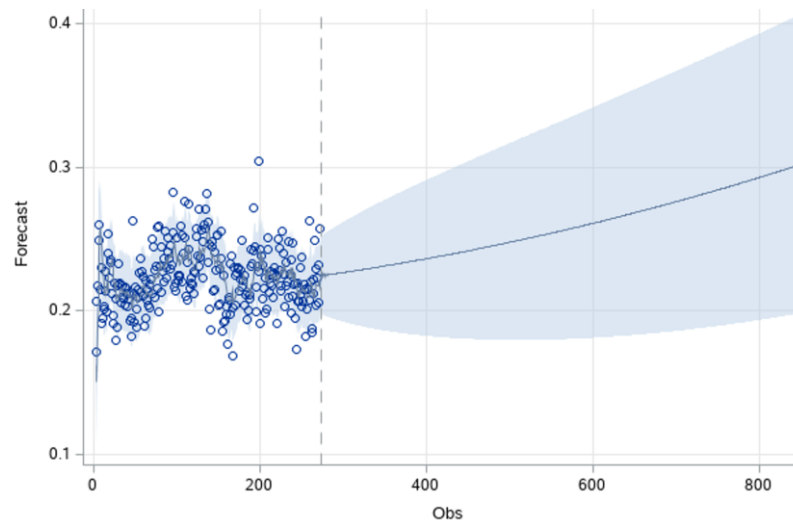


Machine Learning Applied in Rotating Machinery for Anomalies Detection and Remaining Useful Life (RUL) estimation - Case Study



Prepared by:

Ernesto Primera, B.S., M.S.
Machinery Reliability SME
PhD Candidate – University of Delaware

Phone: 713 410 9538
Email: eprimera@udel.edu



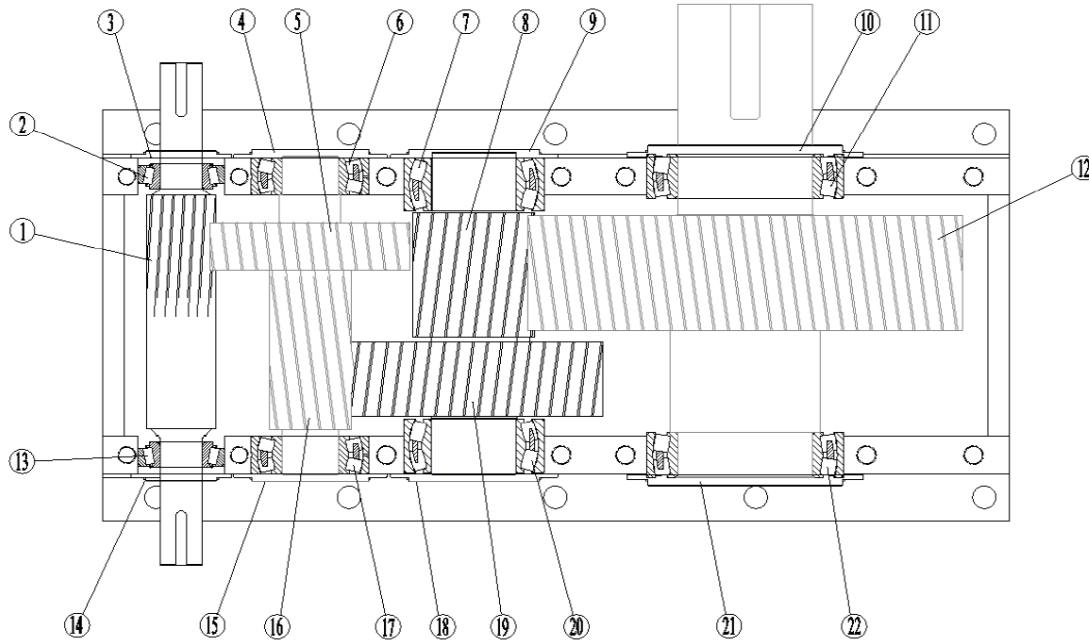
Ernesto Primera

Mechanical/Maintenance Engineer with 24 years of experience in Rotating Machinery, Condition Monitoring, Performance Analysis, and Reliability Evaluations. Experience in the Oil and Gas Industry, Power Plants and OEMs. A passionate about Data Analysis using technology platforms such as: R Studio, SAS, Minitab, SPSS Statistic & Modeler, Risk Simulator, @Risk, MS Power BI, and Tableau. Proven experience as employed for Chevron, Phillips-66, Williams, Flowserve and SKF. During the last 10 years Ernesto have worked in the Rotating Machinery Reliability Group at the Pascagoula Refinery in Mississippi (CHEVRON) and Lake Charles Refinery and Alliance Refinery in Louisiana (PHILLIPS-66). Global Instructor for the American Society of Mechanical Engineers (ASME), Industry Partner and Instructor for the Hydraulic Institute, certified Maintenance & Reliability Professional CMRP, Certified Vibration Analyst Category III by the Technical Associate of Charlotte. Bachelor's Degree in Maintenance Engineering (University Complex AJS - Venezuela), Master's degree in Predictive Maintenance & Diagnostics Technique (Sevilla University - Spain), Master's degree in business Analytics (Grand Canyon University) and currently studying PhD in Applied Statistics in the University of Delaware. Ernesto is currently a SRE Lifetime National Member.

Content

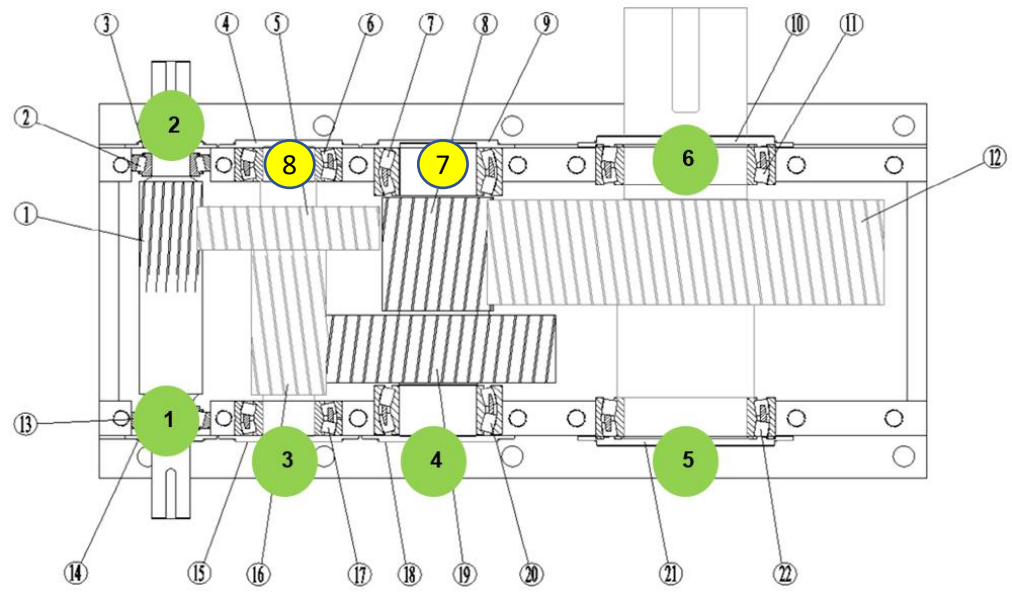
1. Gearbox Technical Info
2. Sensors Hardware Setting
3. Vibration and Temperature Monitoring Software Setting.
4. Vibration Trend Analysis
5. Failure Diagnostic
6. Failure Modes Tags
7. Prognostics of Failure to Identify the Remaining Useful Life.
8. Conclusions.

Technical Info



MODEL	647Z3-S
MO#	96-075332
RATIO	48.48
RATED HP	3300 HP
MOTOR HP	1650 HP
MOTOR RPM	900 RPM
UNIT WEIGHT DRY	53250 LBS
LS COUPLING	18F KOPFLEX
HS COUPLING	FALK 1140T10

Sensors Hardware Setting



The green circles show the place (point) where the Triaxial wireless sensors are installed to measure vibrations and temperature.

- Points # 1 and 2 = 1st Shaft
- Point # 3 = 2nd Shaft
- Point # 4 = 3rd Shaft
- Points # 5 and 6 = 4th Shaft

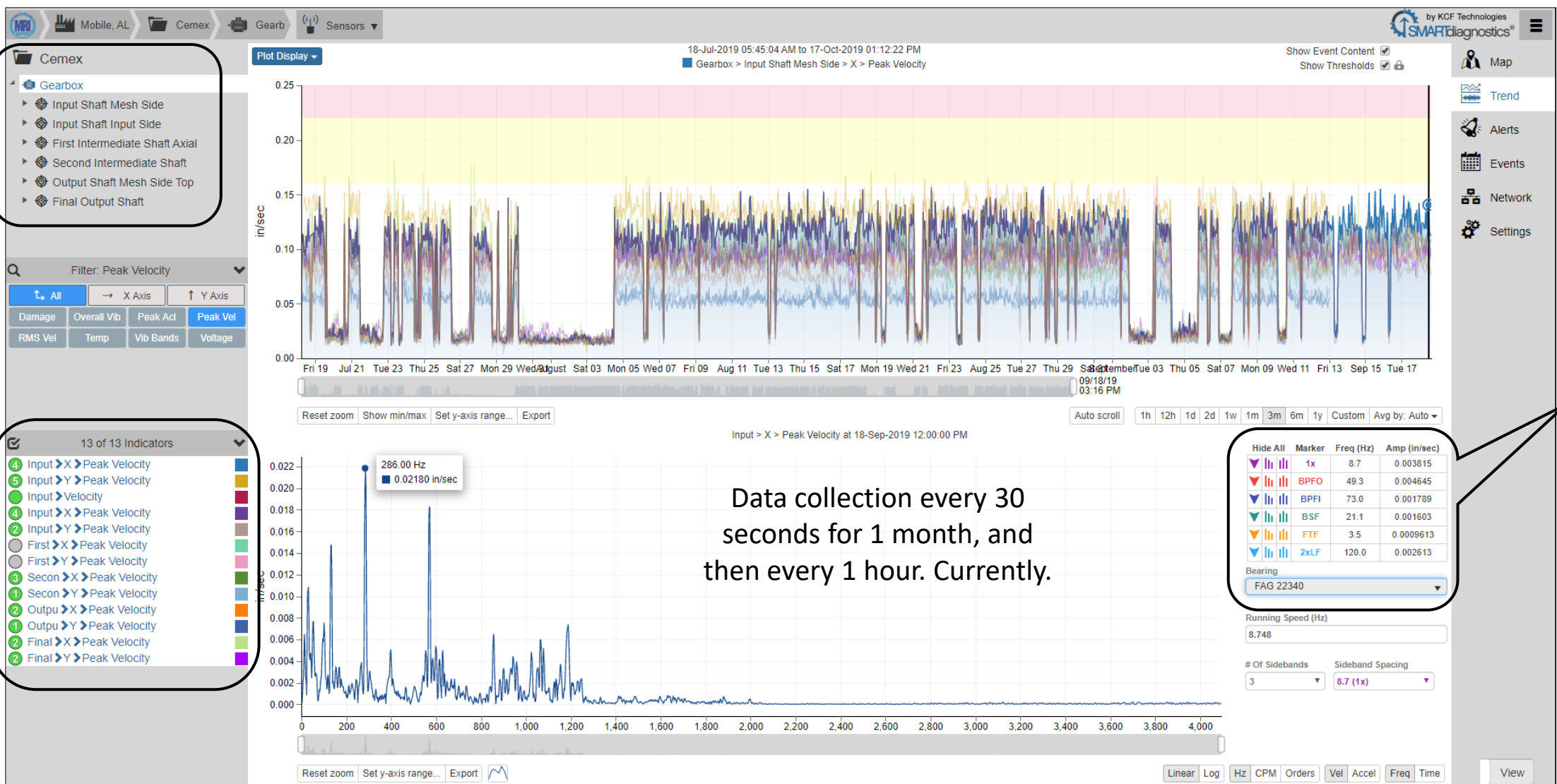


Vibration + Temp. Monitoring Software Setting

Data Points

Data Unit

Bearing Freq.



Data collection every 30 seconds for 1 month, and then every 1 hour. Currently.

Vibration Trend Analysis

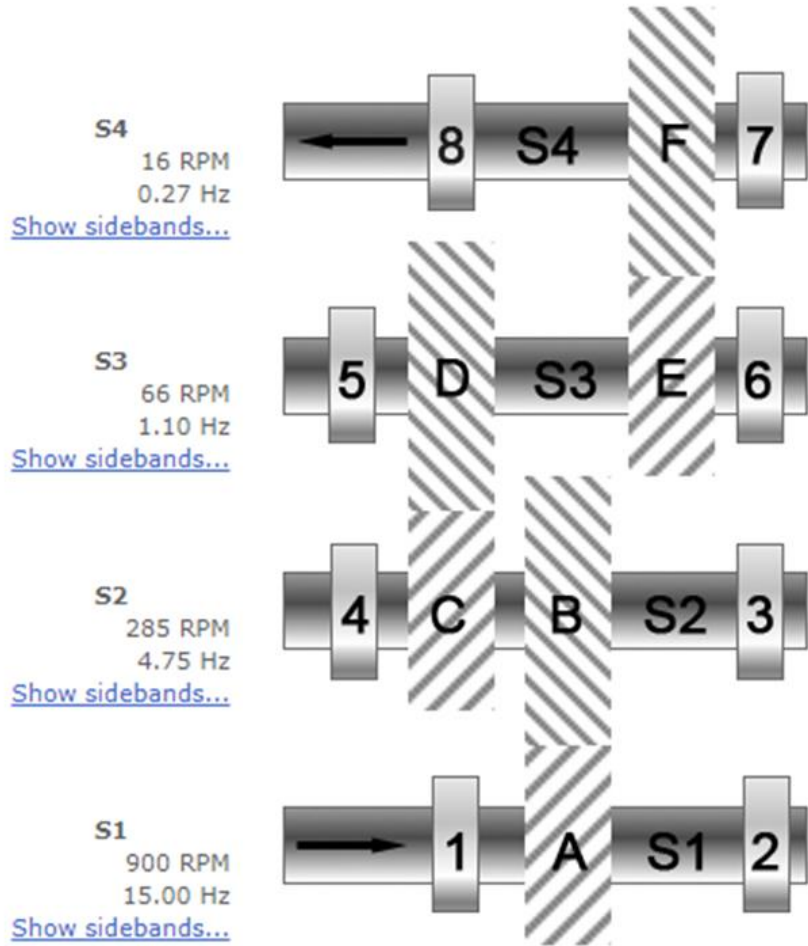
Observations:

The trend chart shows vibration in Velocity values, which are keeping within the alert limits, however there are 6 events in which the global value touches the first alert limit.

Severity Criteria	Color	Description
Green	Green	= Good / Acceptable
Yellow	Yellow	= Alert / In observation
Red	Red	= Warning / Need Attention



Failure Diagnostic Gears Frequencies Analysis



S4 E & F		
	CPM	Hz
-2	899	14.98
-1	915	15.25
0	931	15.52
1	947	15.78
2	963	16.05

A complete simulation of the Gearbox train was performed, to identify the specific failure frequencies for each shaft and each gear.

Shaft	Speed	Gear Teeth	GMF	GMF Orders				MA Ratio	
				S1	S2	S3	S4	S1	S4
S1	900 RPM								
	15.00 Hz	A 19	A & B	17100 CPM	19.0	60.0	257.1	1083.7	
S2	285 RPM	B 60		285.00 Hz					S2 0.317
	4.75 Hz	C 14	C & D	3990 CPM	4.4	14.0	60.0	252.9	
S3	66 RPM	D 60		66.50 Hz					S3 0.074
	1.10 Hz	E 14	E & F	931 CPM	1.0	3.3	14.0	59.0	
S4	16 RPM	F 59		15.52 Hz					S4 0.018
	0.27 Hz								

Failure Diagnostic

Bearings Frequencies Analysis

Bearing Frequencies Calculator

This calculator is used to calculate the different bearing defect frequencies of bearing applications. One can search for an existing SKF bearing or input the different bearing parameters manually. The bearing defect frequencies can be displayed in Hertz, CPM or in orders of the rotational speed.

Bearing Data

- SKF bearing designation*: 22340 CC/W33
- Measurement system: Metric Imperial
- Bearing type*: SRB
- Pitch diameter*: 12.33 in
- Rolling element diameter*: 2.205 in
- Number of rolling elements (per row)*: 15
- Contact angle*: 12.75 degrees
- Rotational speed*: 285 rpm
- Rotating ring*: inner outer

Output

Hertz CPM Orders

Shaft speed frequency	4.750 Hz
Inner race defect frequency (BPFI)	41.839 Hz
Outer race defect frequency (BPFO)	29.411 Hz
Cage defect frequency (FTF)	1.961 Hz
Ball spin frequency (BSF)	12.877 Hz
Rolling element defect frequency	25.753 Hz

Source: SKF.

Bearing Frequencies Calculator

This calculator is used to calculate the different bearing defect frequencies of bearing applications. One can search for an existing SKF bearing or input the different bearing parameters manually. The bearing defect frequencies can be displayed in Hertz, CPM or in orders of the rotational speed.

Bearing Data

- SKF bearing designation*: 24160 CC/W33
- Measurement system: Metric Imperial
- Bearing type*: SRB
- Pitch diameter*: 15.84 in
- Rolling element diameter*: 1.969 in
- Number of rolling elements (per row)*: 22
- Contact angle*: 14.75 degrees
- Rotational speed*: 66.58 rpm
- Rotating ring*: inner outer

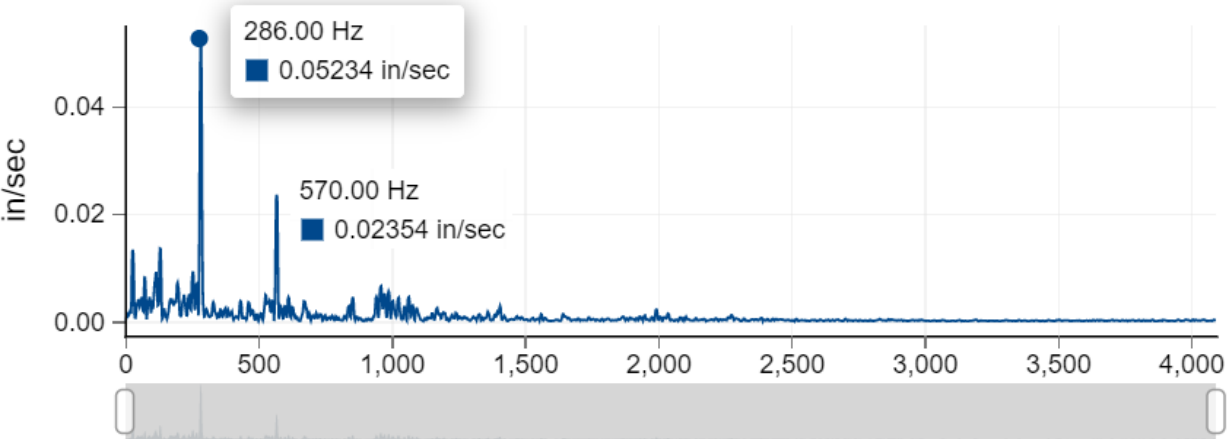
Output

Hertz CPM Orders

Shaft speed frequency	1.110 Hz
Inner race defect frequency (BPFI)	13.674 Hz
Outer race defect frequency (BPFO)	10.739 Hz
Cage defect frequency (FTF)	0.488 Hz
Ball spin frequency (BSF)	4.399 Hz
Rolling element defect frequency	8.798 Hz

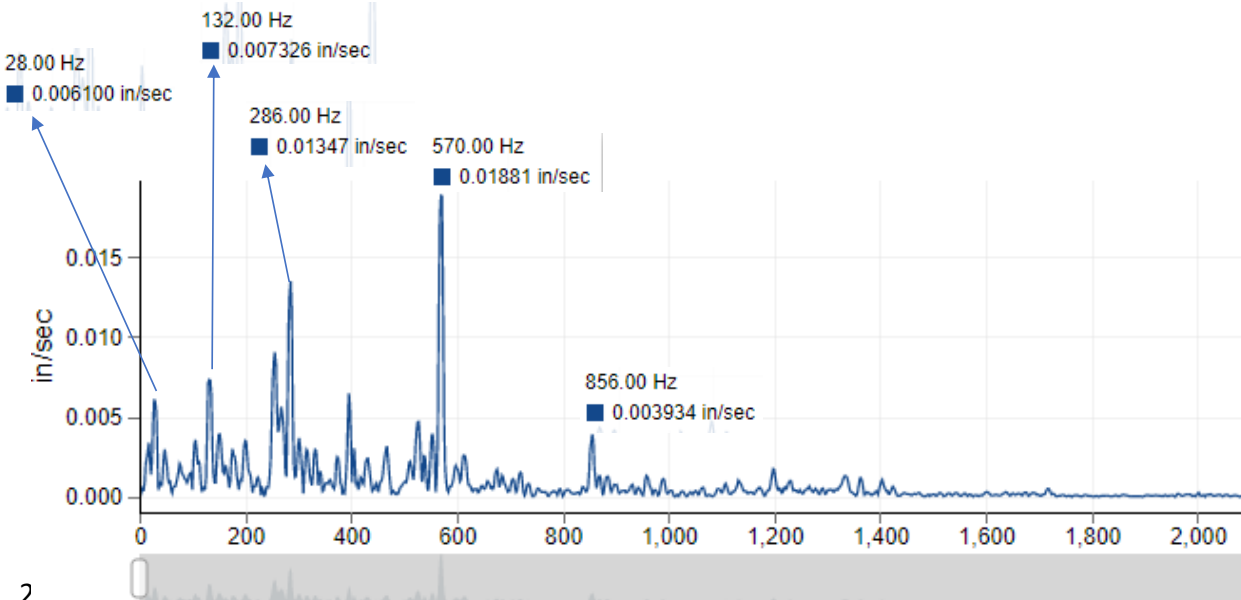
Failure Diagnostic

Input > Y > Peak Velocity at 12-Jul-2019 11:24:04 AM



Spectrum Analysis Shaft N°1

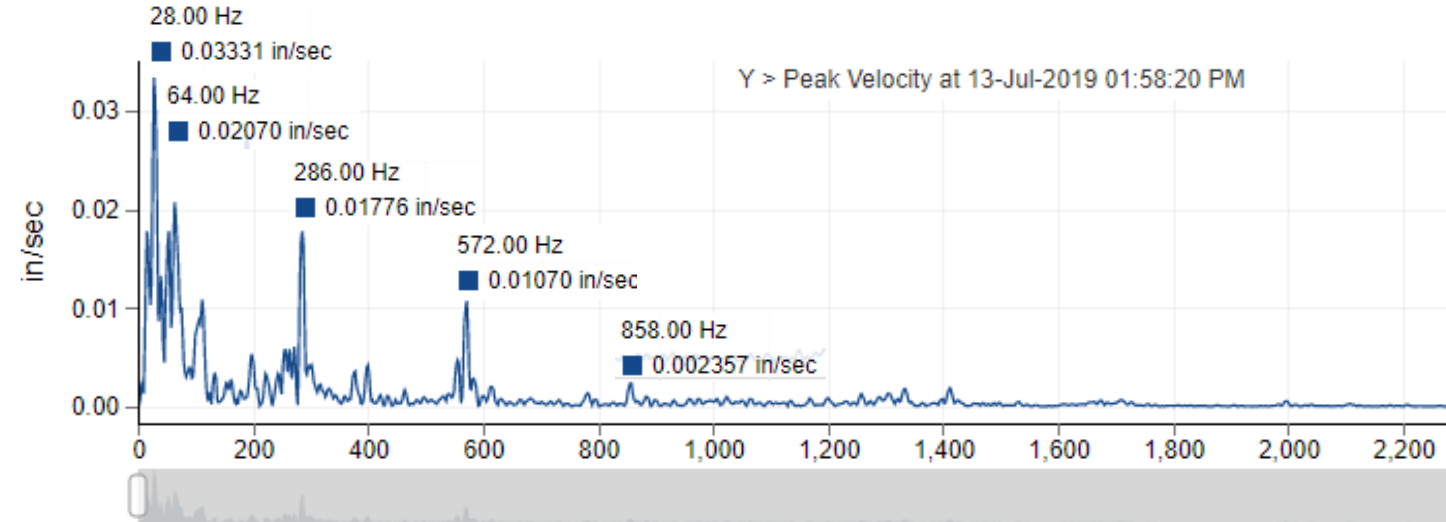
The shaft 1 clearly shows the failure frequencies of gear teeth that are highlighted in the simulation, these correspond to 1x and 2x of the frequency of failure of the gears of shaft 2, which are transmitted to shaft 1.



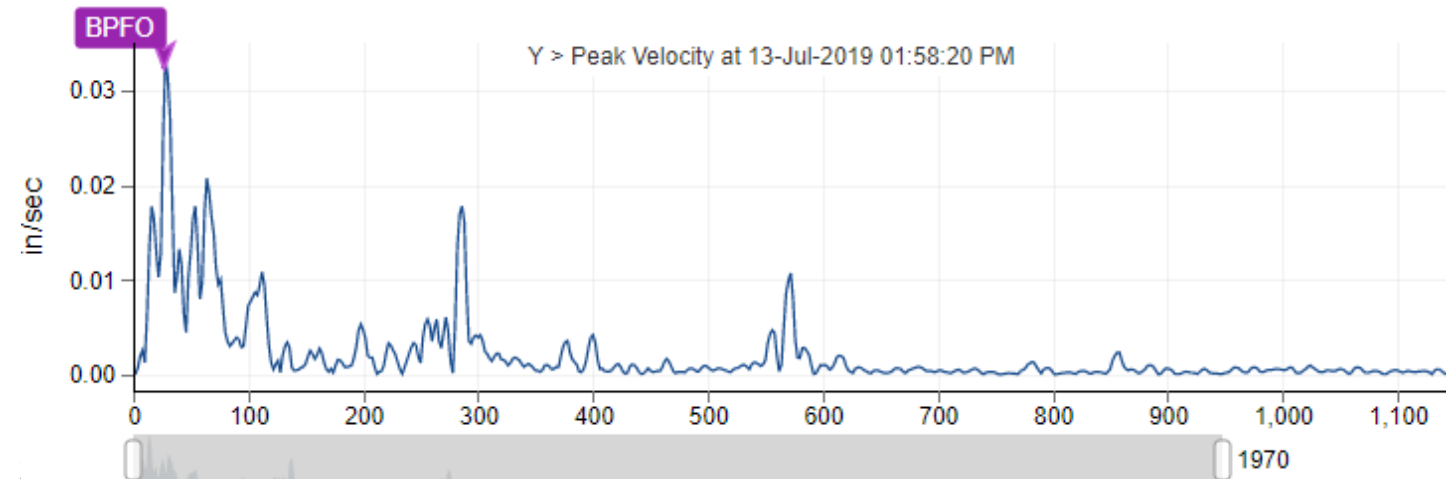
With a frequency amplification, we can also observe the coupling failure frequency (28 Hz), as well as the appearance of the 3x (856 Hz) of the gear teeth failure of shaft 2.

Failure Diagnostic

Spectrum Analysis Shaft N°2



The shaft 2 clearly shows the frequencies 1x, 2x and 3x of the gear tooth failure frequency, as well as a high amplitude in the coupling failure frequency (28 Hz) that is amplified with the BPFO frequency (Ball Pass Frequency Outer) corresponding to the shaft 2 bearing.

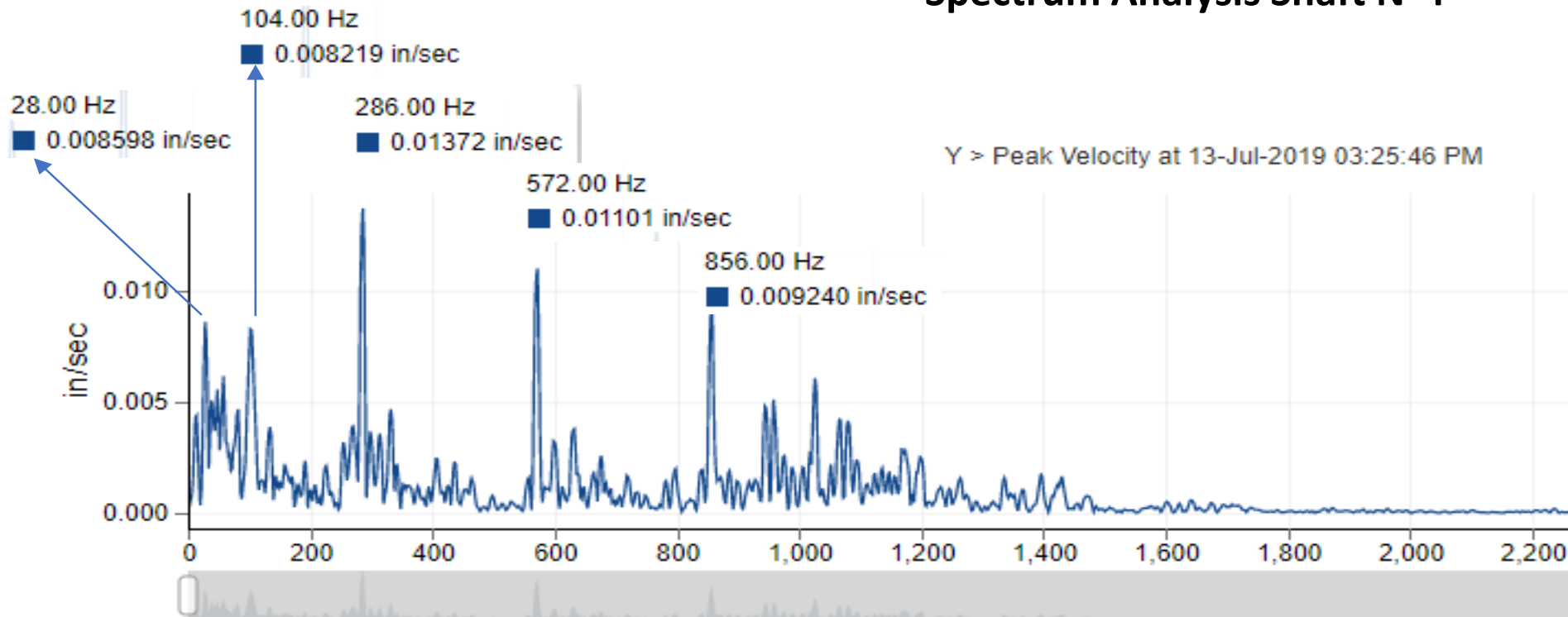


Failure Diagnostic

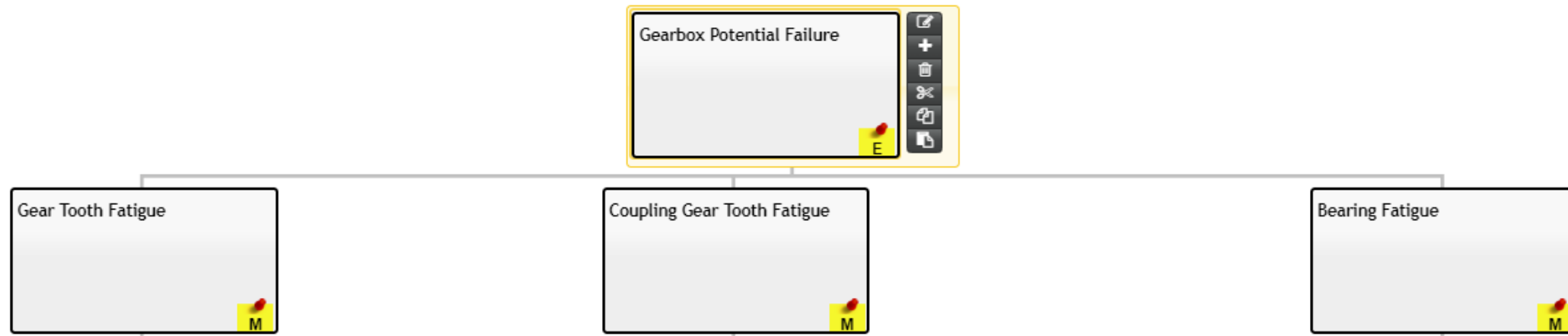
The gear coupling failure frequency and the shaft 2 gear tooth failure frequencies are shown again.

In the different spectral graphs, the frequency of the shaft 3 gear is also observed, which is the other frequency highlighted in the gearbox train simulation.

Spectrum Analysis Shaft N°4



Failure Modes Tags

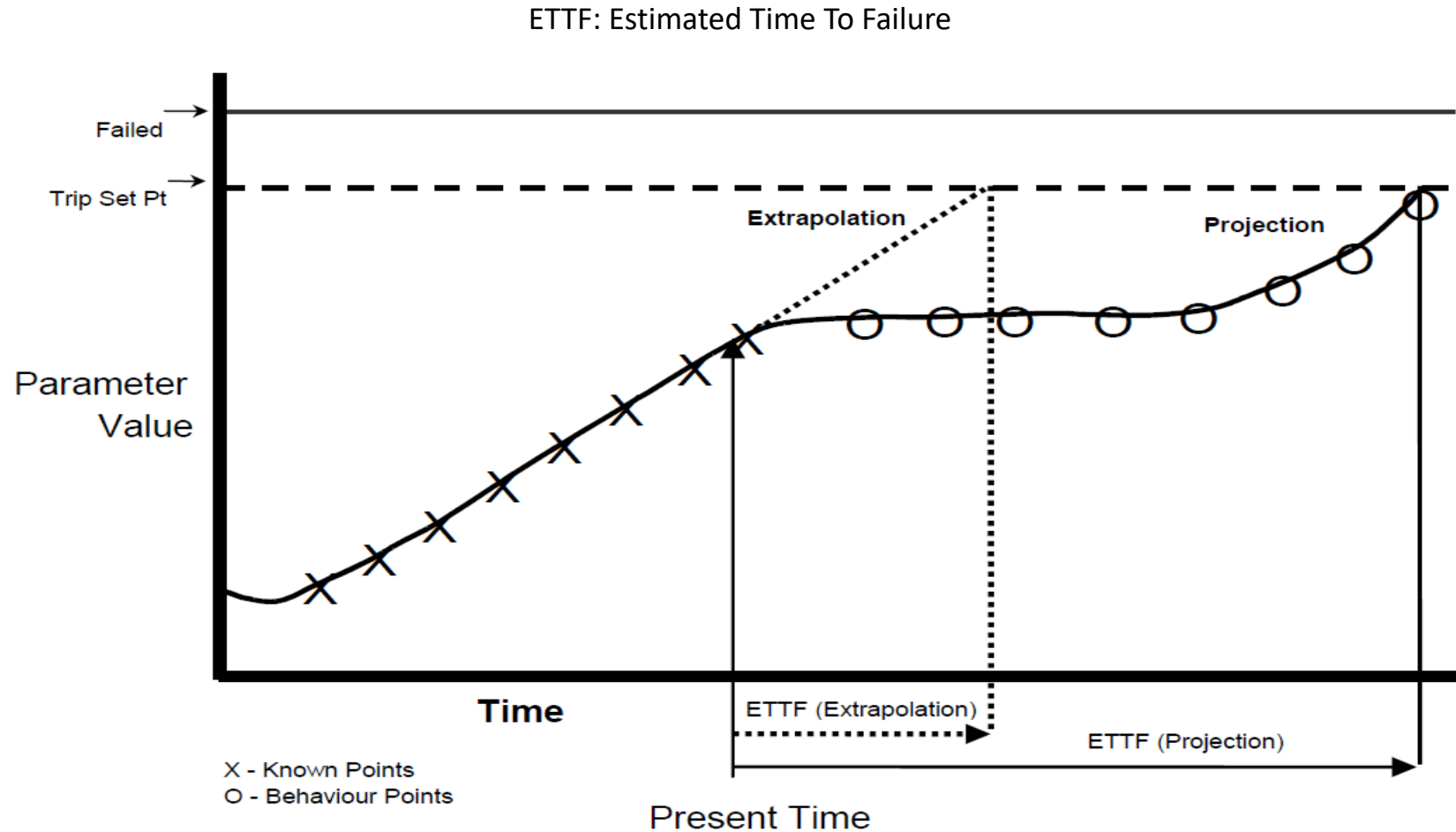


Three (3) failure modes identified:

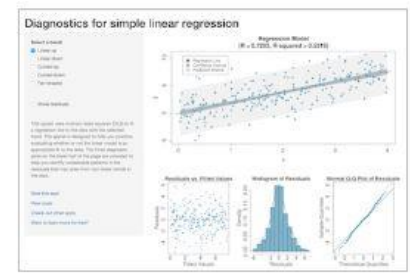
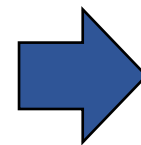
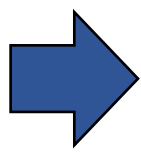
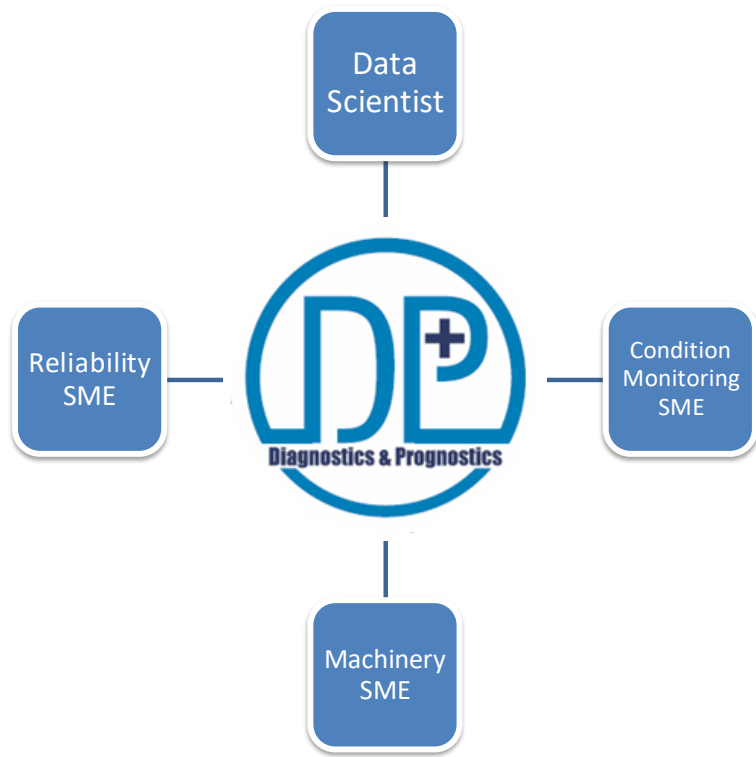
Failure Modes:

- **Gear Coupling Fatigue.**
- **Gears Shaft Fatigue. (Shaft 2 and 3)**
- **SKF 22340 Bearing Fatigue. (Shaft 2).** This is the smallest and most susceptible bearing in the machinery train and is an effect of the inadequate clearance of Bearing TIMKEN HH234048 in the Shaft N°1.

Prognostics of Failure to Identify the Remaining Useful Life

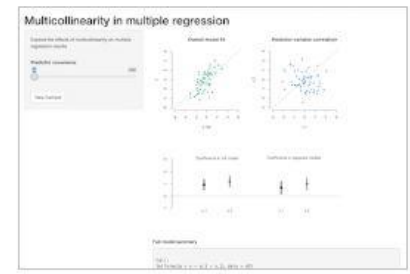


Prognostics of Failure to Identify the Remaining Useful Life



LINEAR REGRESSION DIAGNOSTICS

Explore the fit of a linear regression.

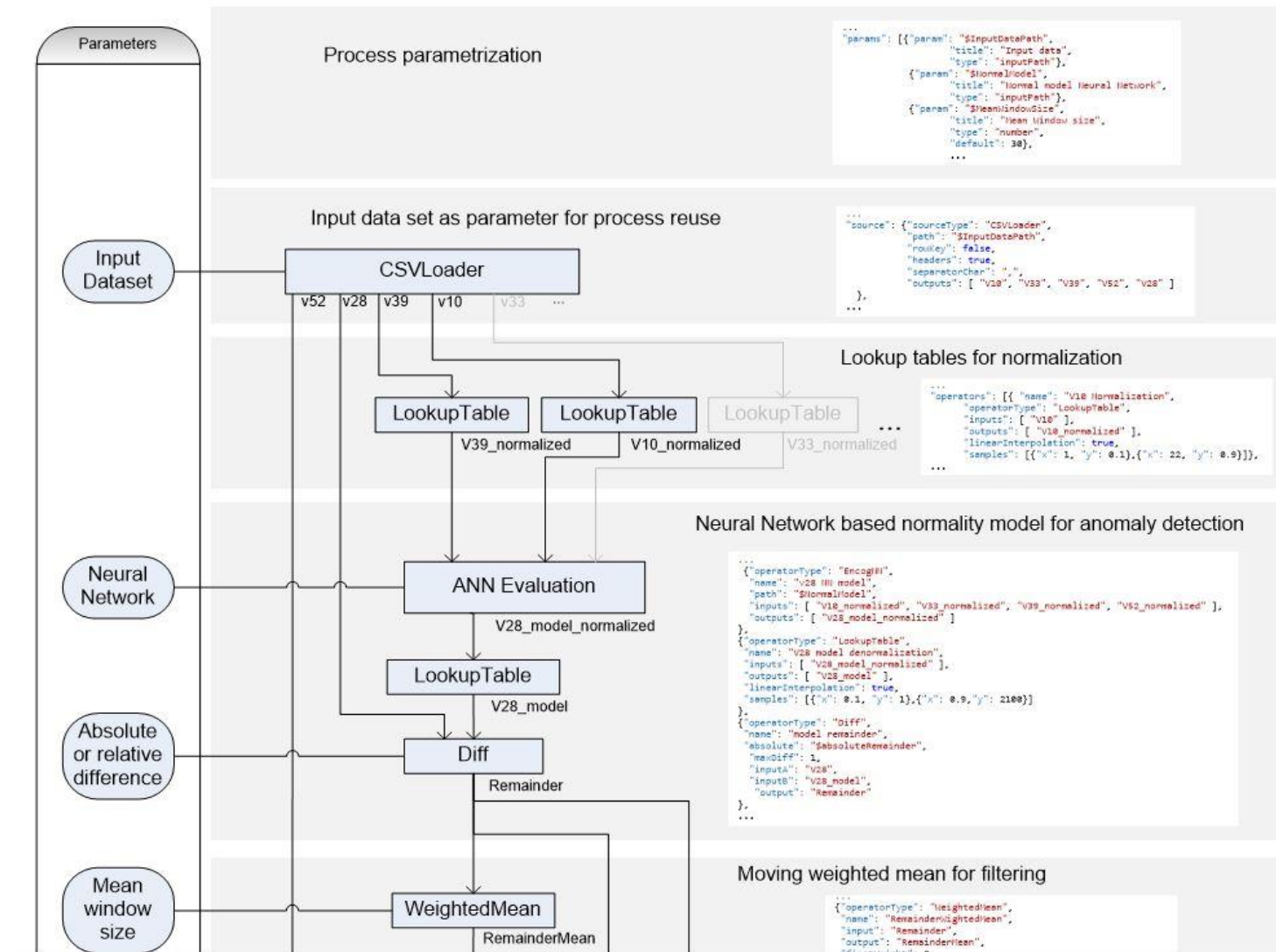


MULTICOLLINEARITY

Simulate correlation between model predictors.

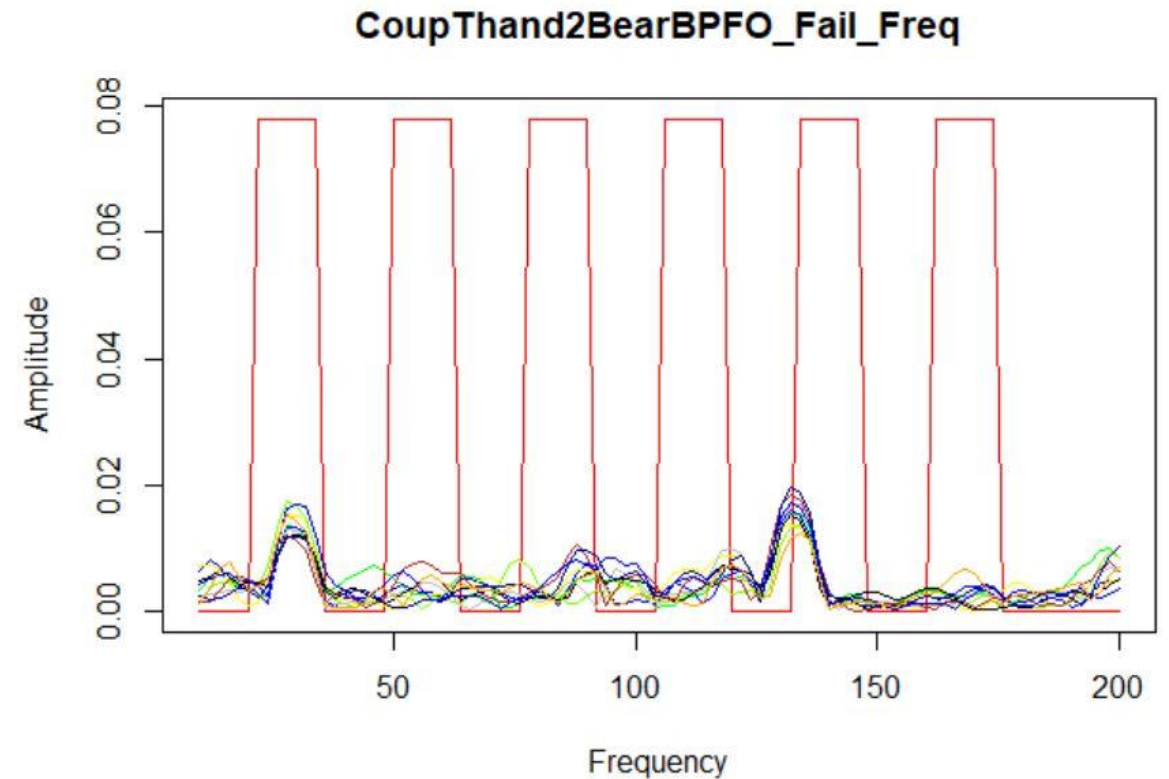
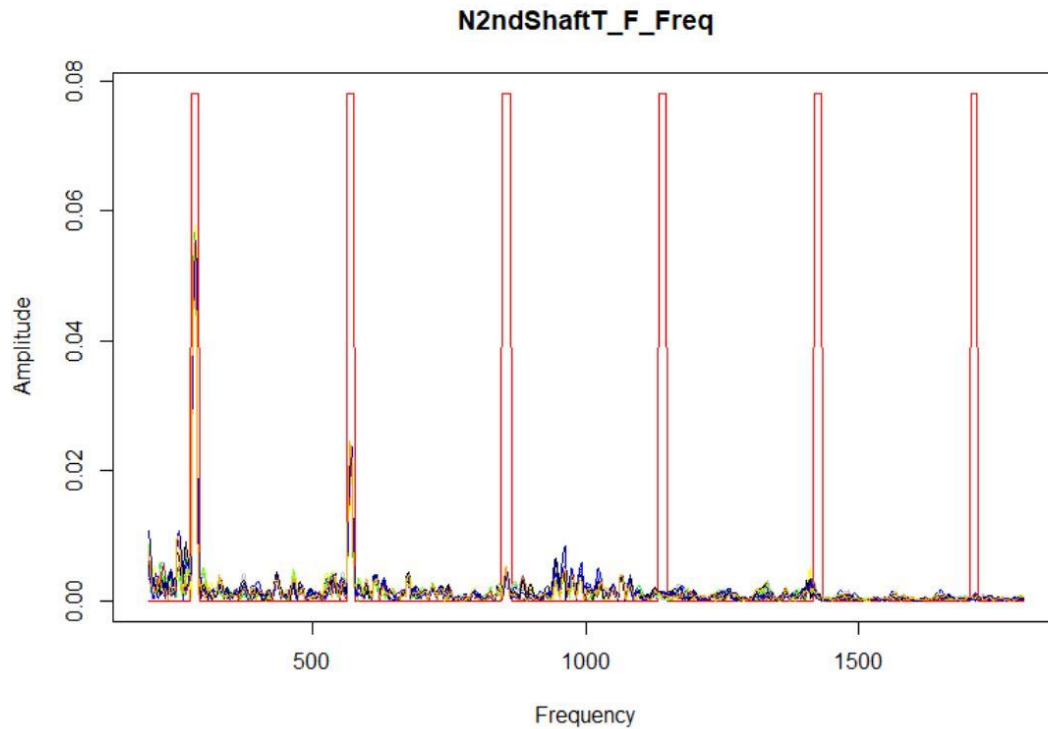
Prognostics of Failure to Identify the Remaining Useful Life

Model and Programming



Prognostics of Failure to Identify the Remaining Useful Life

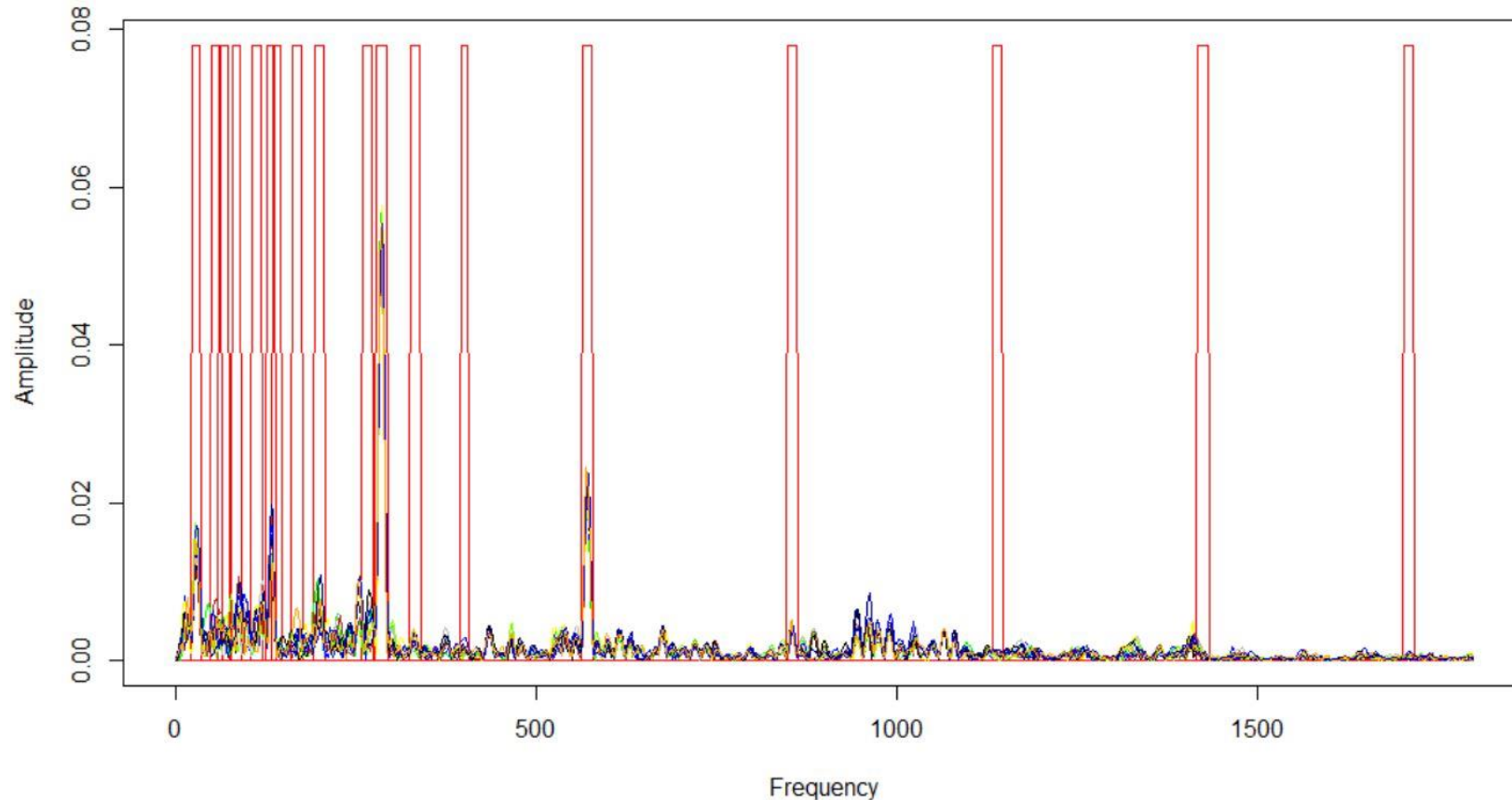
Spectrum Prognosis focused on Failure Modes Tags



Prognostics of Failure to Identify the Remaining Useful Life

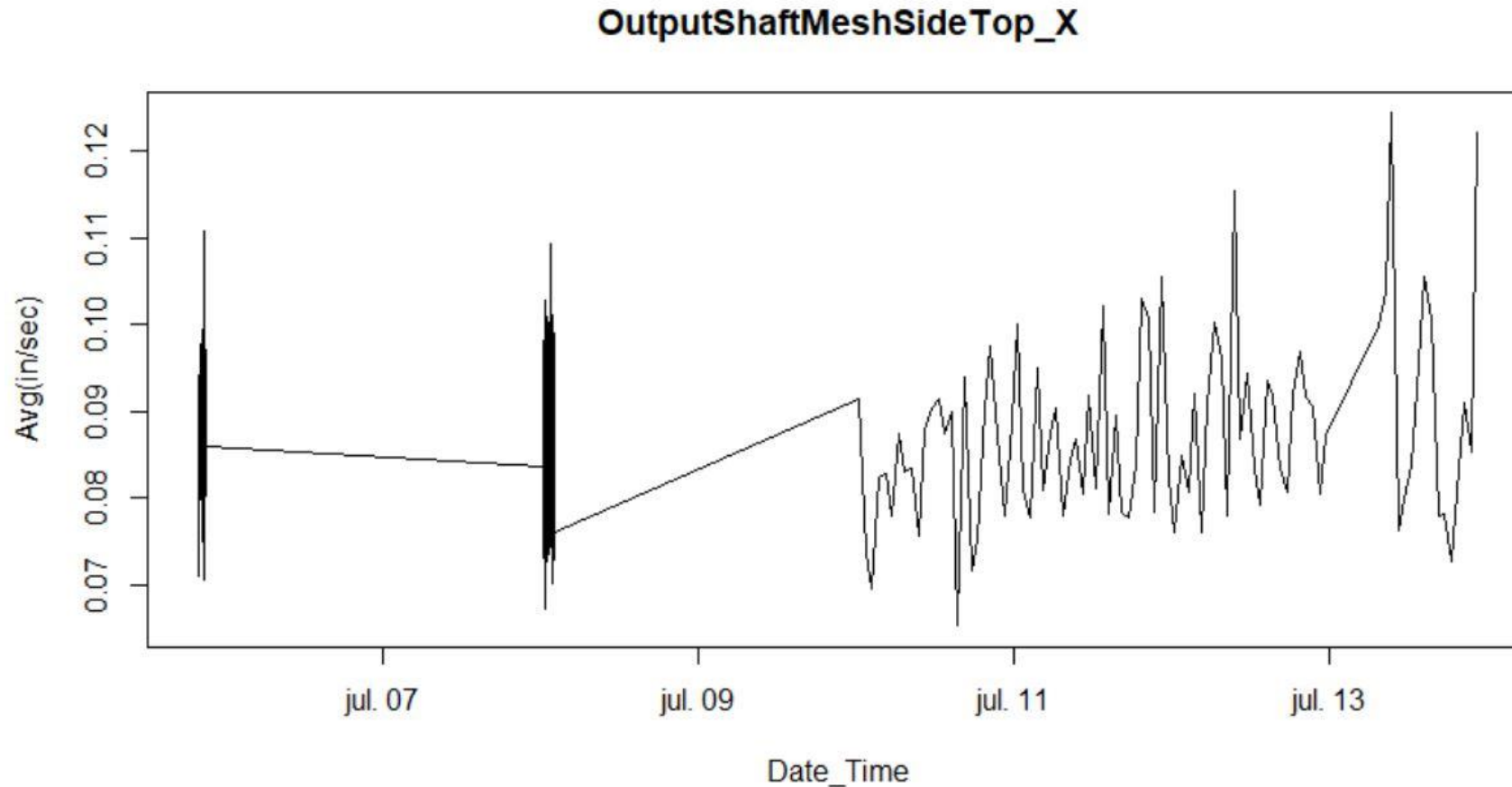
Spectrum Prognosis focused on Failure Modes Tags

Data(colors) and Failure Frequencies(FF,red) only for FF



Prognostics of Failure to Identify the Remaining Useful Life

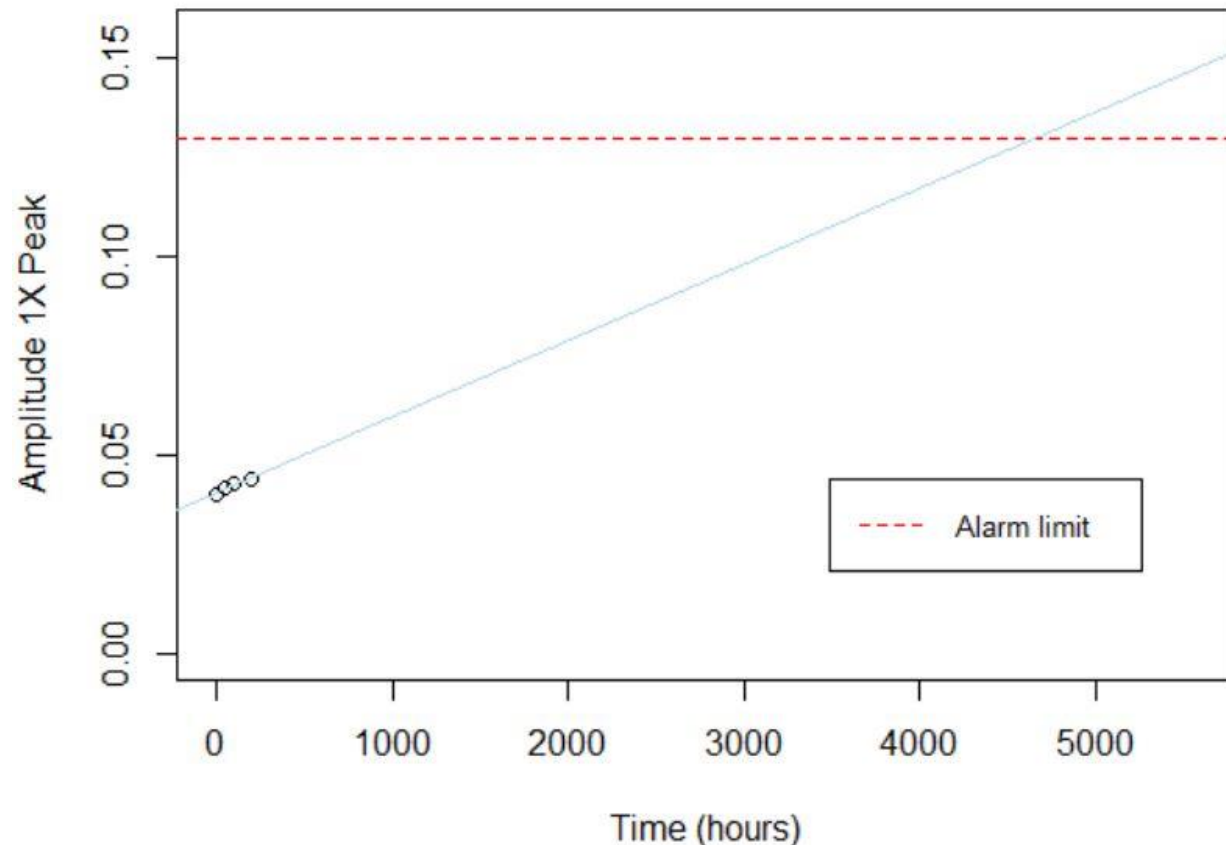
Gear Mech Projection



Prognostics of Failure to Identify the Remaining Useful Life

Remaining Useful Life (RUL)

GearBox Alarm Level prediction



Conclusion:

The degradation process is slow, however the evolution of the 1X spectra allows to predict, with low level of confidence because we have very few spectrums and very close in time to each other, that around 1X the maximum amplitude of Failure frequency **will cross the alarm limit in 4647 hours**. There is not enough data to use Neural Networks, so we use linear regression.

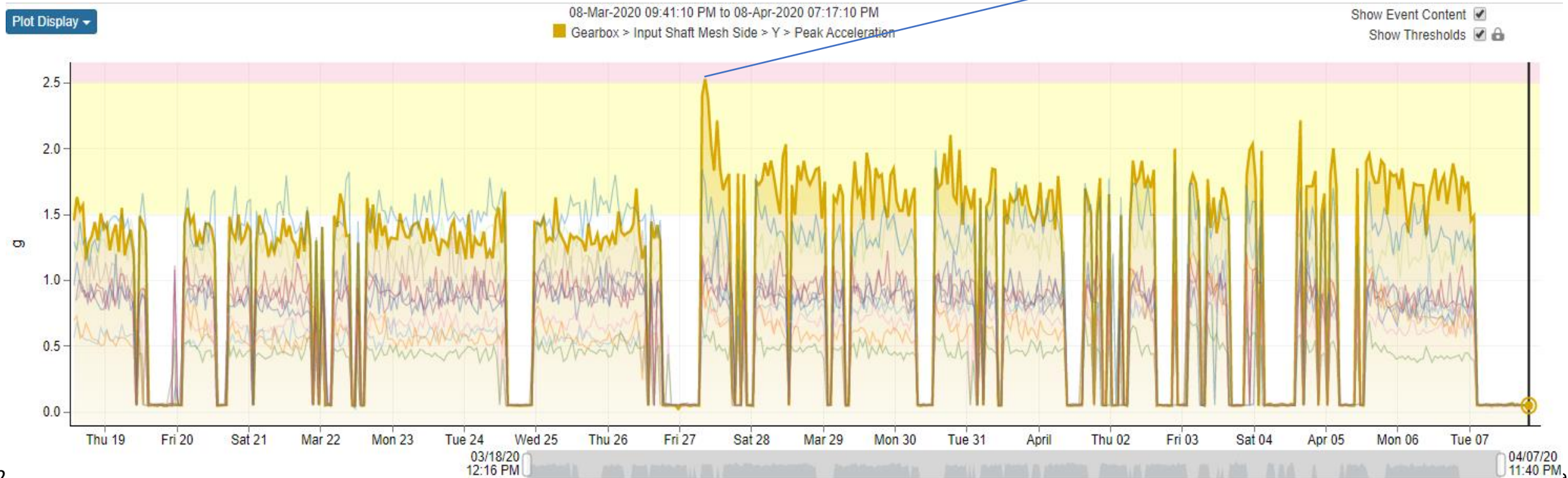
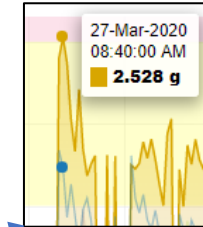
(Approximately **March-April 2020**)

Prognostics of Failure to Identify the Remaining Useful Life

Remaining Useful Life (RUL)

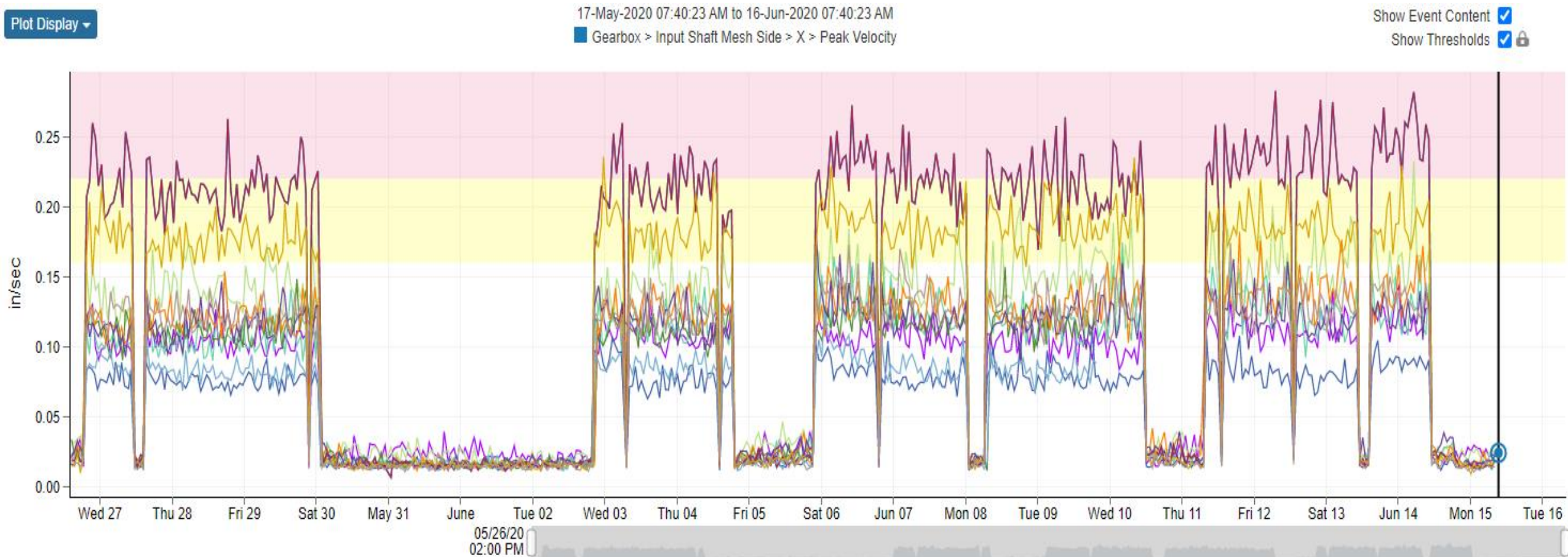
Observations:

The trend chart shows vibration in Velocity values, which are keeping within the alert limits.



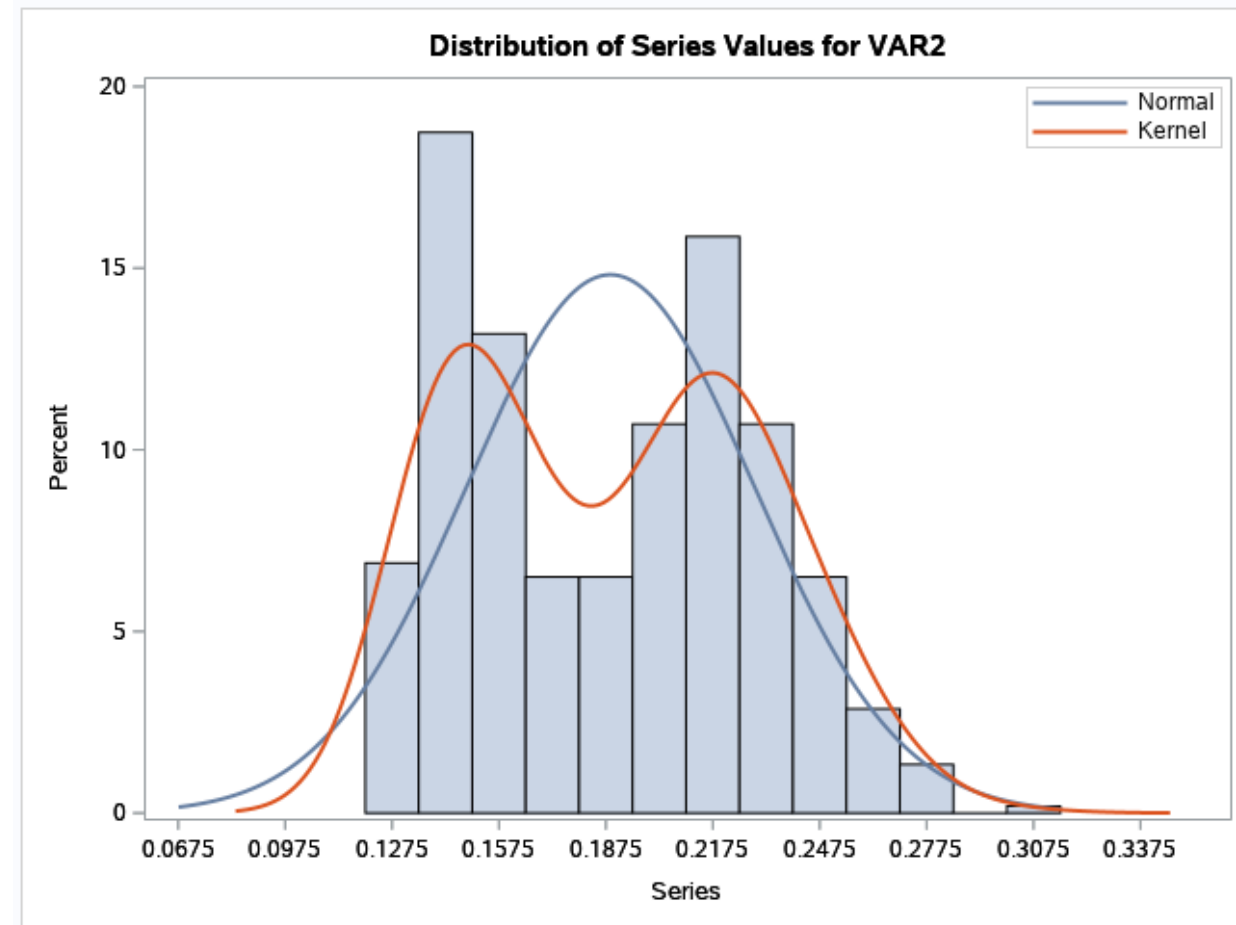
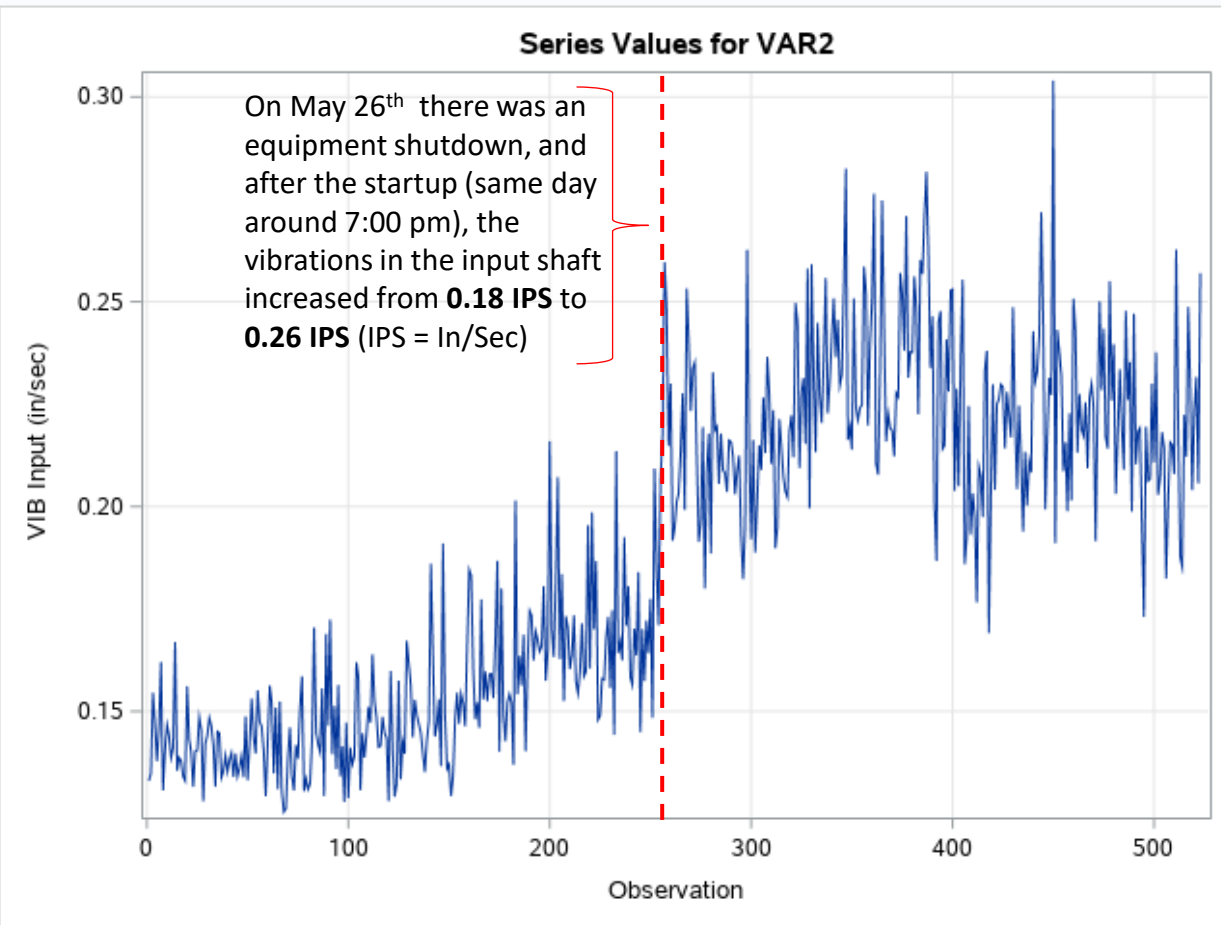
Prognostics of Failure to Identify the Remaining Useful Life

Remaining Useful Life (RUL)



Observations:

1. Two cycles are clearly observed. Before and after the event that occurred on May 26th where the vibrations increased.
2. The two cycles are clearly observed on the orange curve in the frequency distribution histogram as well.
3. Based on what is described in the two previous paragraphs, the forecast will be made only with the data of the second cycle, which begins on May 26th.



Prognostics of Failure to Identify the Remaining Useful Life

Vibration Trend:

Variables:

- Input Shaft
- Output Shaft

Vibration Unit:

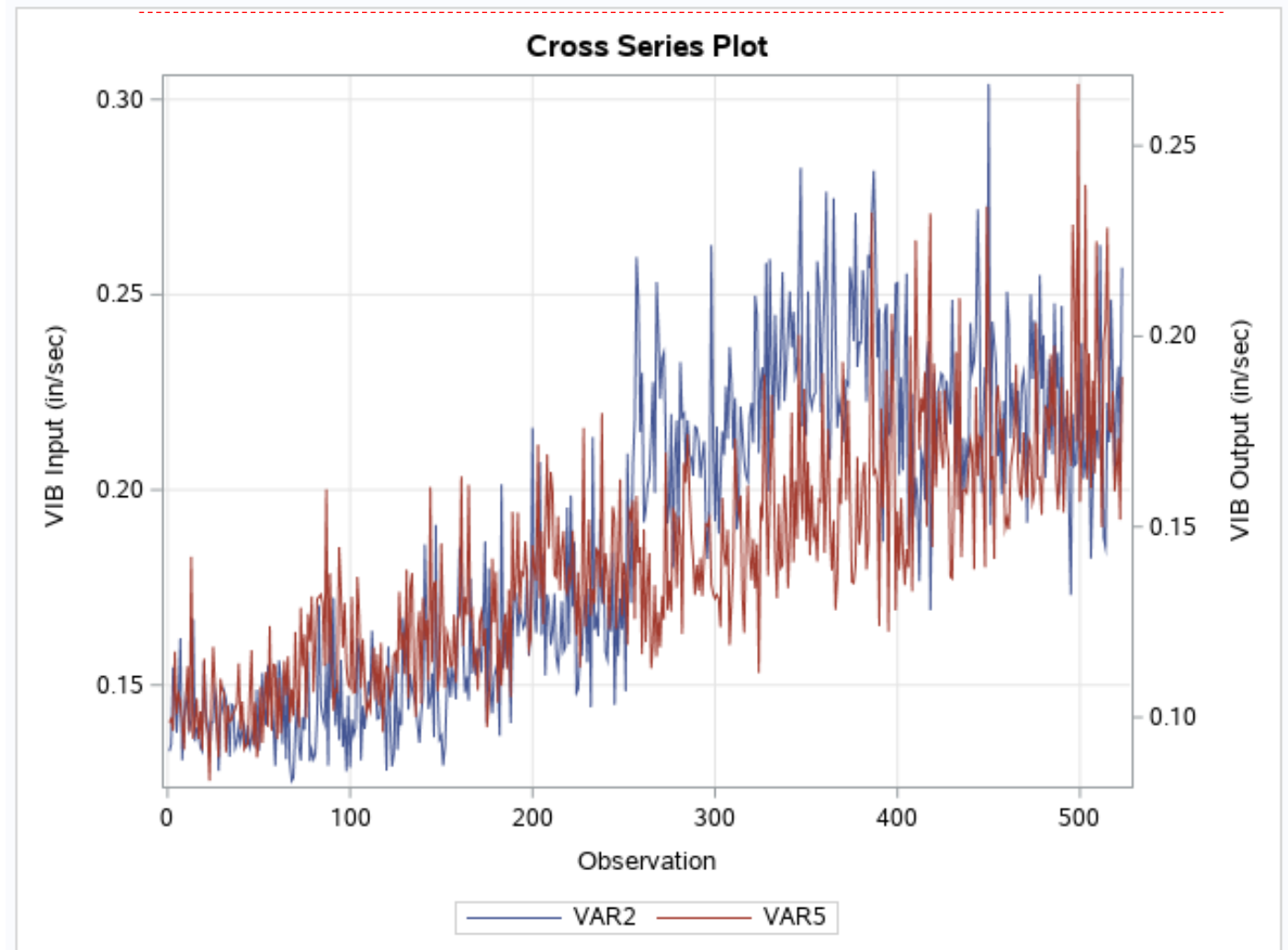
In/Sec

Vibration Alarm Limit:

0.30 in/Sec

Timeframe:

July 2019 to July 2020

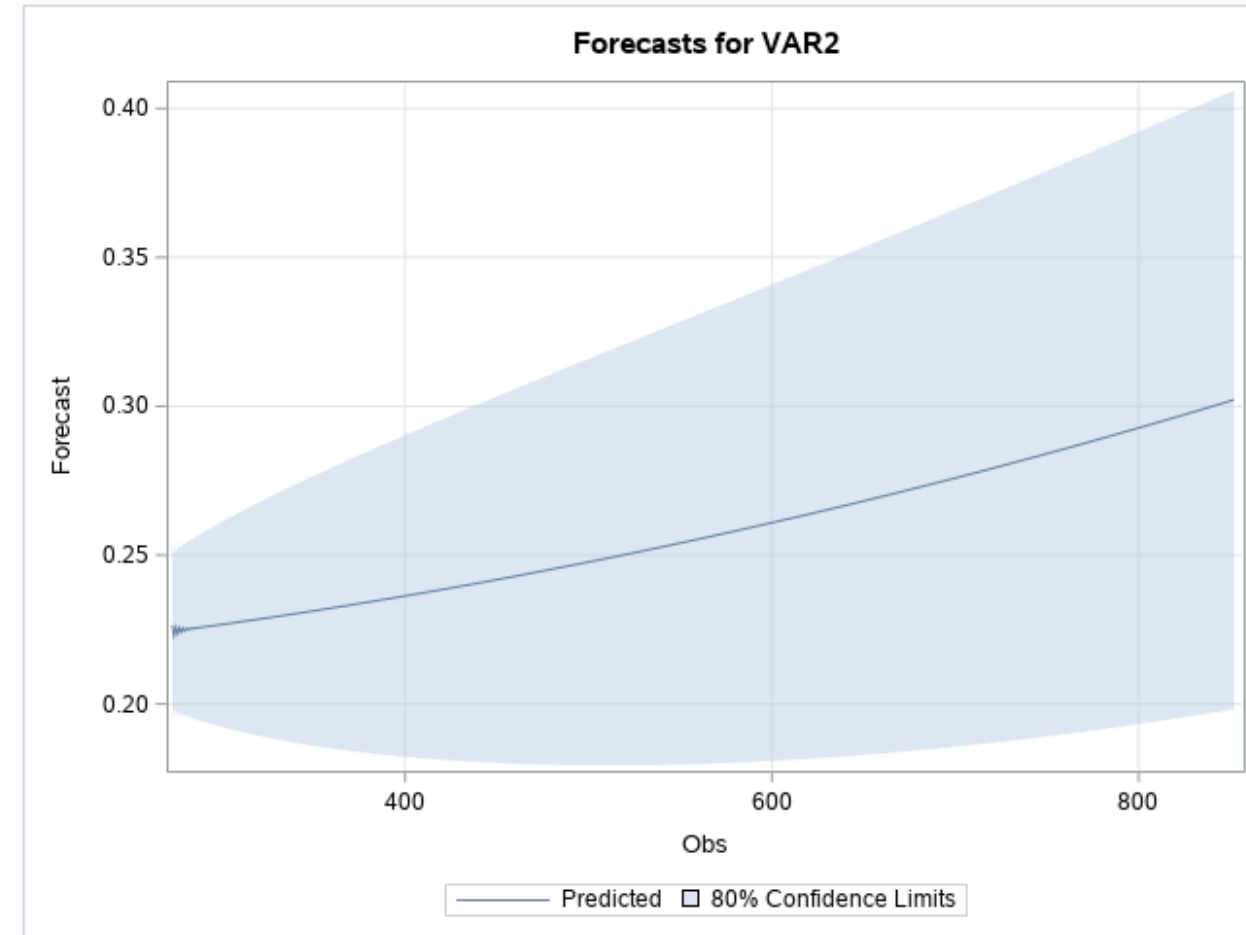
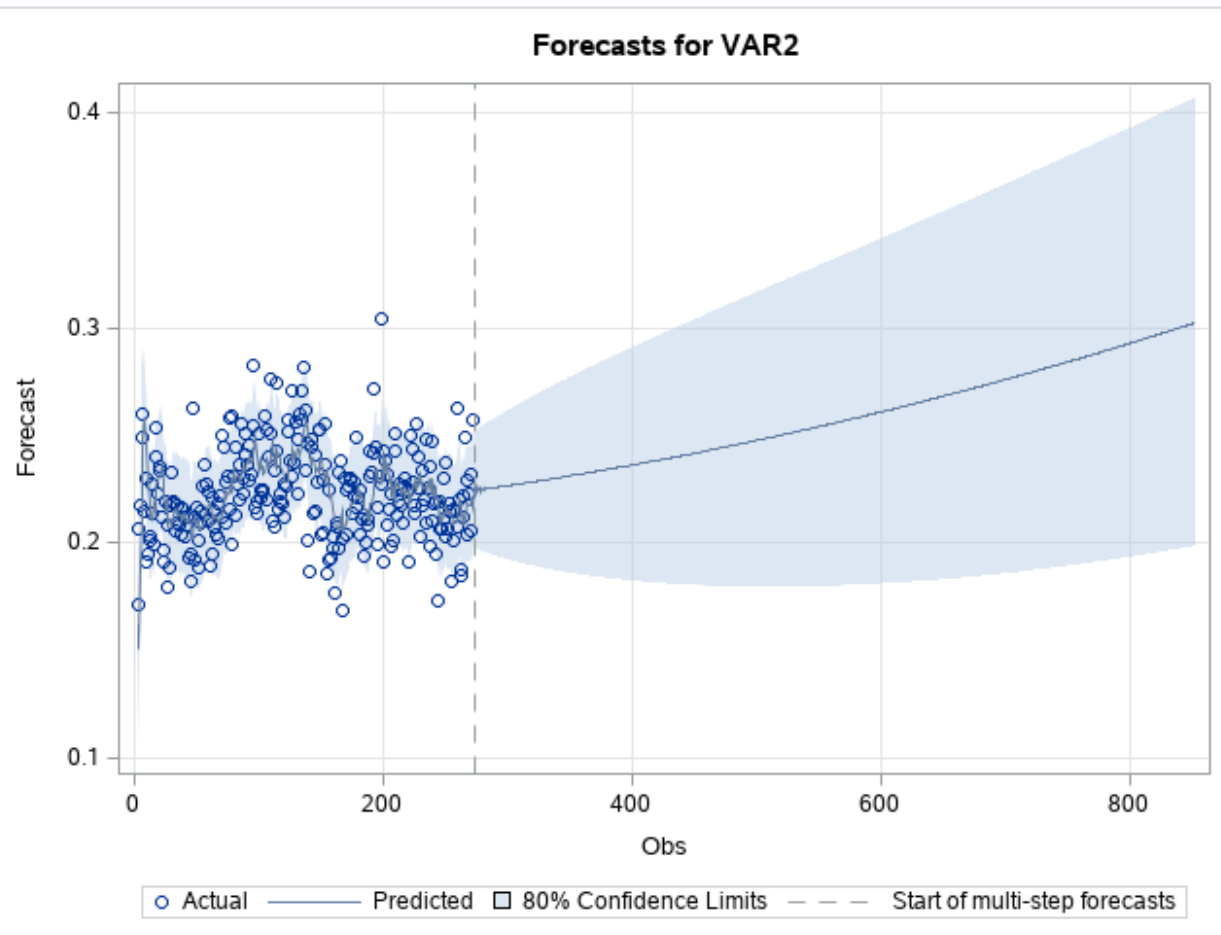


Prognostics of Failure to Identify the Remaining Useful Life

Summary:

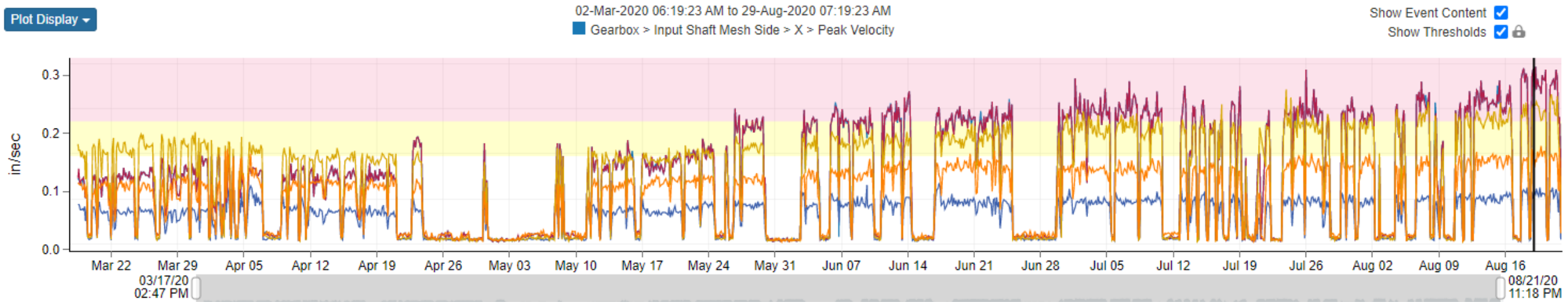
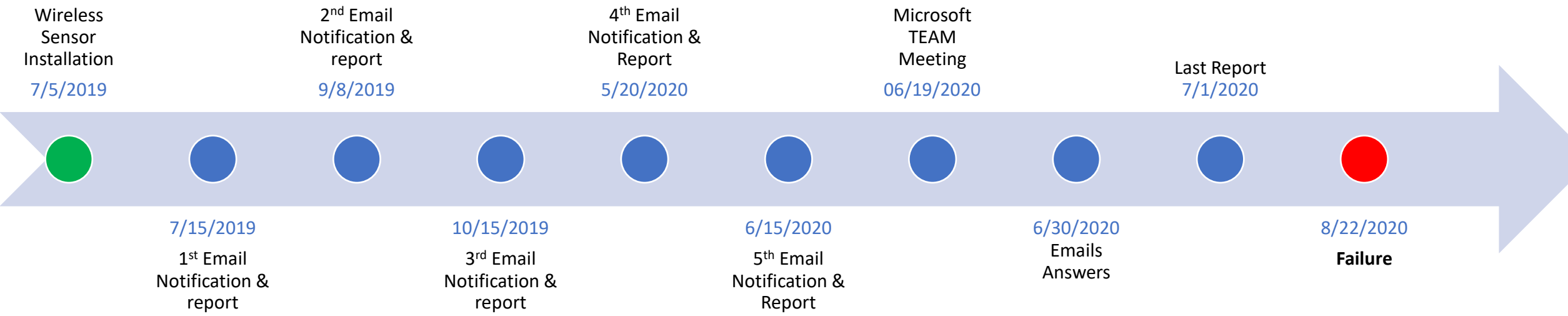
The observations (Obs) are carried out every 1 hour, for that reason 950 Obs are approximately 950 Hours \approx 40 Days.

Vibration Prognosis \geq 0.40 IPS \approx August 9, 2020.



Prognostics of Failure to Identify the Remaining Useful Life

Timeline



Conclusions

Three (3) failure modes and two (2) potential causes are identified:

Failure Modes:

- Gear Coupling Fatigue.
- Gears Shaft Fatigue. (Shaft 2 and 3)
- Bearing Fatigue. (Shaft 2)

Root Causes:

- An inadequate bearing clearance (Shaft 1) has created gear overload. (validated)
- Improper assembly and installation of the coupling has caused the coupling to fail, and the Gear Coupling Failure has created dynamic loads on the gears train.

The case shows the success of applying advanced models and algorithms for equipment failure diagnosis and prognosis with time series variables such as vibrations. It was essential to have multiple vibrational variables through the shaft train of the gearbox, as well as the identification of failure modes to ensure their observability. One of the keys of the model was the application of identification of patterns of vibration frequencies in the vibrational spectrum and the projection of the amplitude of these frequency peaks through the regression model, which shows a straightforward combination of traditional knowledge about vibration analysis, and the application of data science models.